

Supplement to: Improved representation of laminar and turbulent sheet flow in subglacial drainage models

Tim Hill¹, Gwenn E. Flowers¹, Matthew J. Hoffman², Derek Bingham³, Mauro A. Werder^{4,5}

¹*Department of Earth Sciences, Simon Fraser University, Burnaby, BC, Canada*

²*Fluid Dynamics and Solid Mechanics Group, Los Alamos National Laboratory, Los Alamos, NM, USA*

³*Department of Statistics and Actuarial Science, Simon Fraser University, Burnaby, BC, Canada*

⁴*Laboratory of Hydraulics, Hydrology and Glaciology (VAW), ETH Zurich, Zurich, Switzerland*

⁵*Swiss Federal Institute for Forest, Snow and Landscape Research (WSL), Birmensdorf, Switzerland*

Correspondence: Tim Hill <tim_hill_2@sfu.ca>

S1 METHODS

S1.1 Model parameters

To apply GlaDS to a synthetic Greenlandic catchment with seasonally varying melt forcing, key model parameters (bed bump height h_r and wavelength l_r , the width of sheet flow contributing to channel discharge l_c , and channel conductivity k_c) are increased in magnitude to accommodate the larger scale of the Greenland catchment compared to an alpine glacier catchment and the transient, surface input-dominated drainage characteristic of Greenland rather than steady Antarctic hydrology (Table 1). We envision the controlling bed bump scale for subglacial drainage beneath the Greenland Ice Sheet to be larger (bed bump length $l_r = 10$ m and height $h_r = 0.5$ m) than is often used for alpine glacier hydrology (e.g., $l_r = 2$ m, $h_r = 0.1$ m for Gornergletscher; Werder and others, 2013) or steady state Antarctic hydrology (e.g., $l_r = 2$ – 4 m, $h_r = 0.08$ – 0.1 m; Dow and others, 2022; Hager and others, 2022). This bedrock bump scale is smaller than that used by Cook and others (2022) for transient simulations of Store Glacier ($l_r = 100$ m, $h_r = 1$ m) and larger than that of Downs and others (2018) for a synthetic Greenland catchment ($l_r = 2$ m, $h_r = 0.1$ m). The channel conductivity $k_c = 0.5 \text{ m}^{3/2} \text{ kg}^{-1/2}$ is slightly larger than that of Cook and others (2022) and Downs and others (2018) ($k_c = 0.1 \text{ m}^{3/2} \text{ kg}^{-1/2}$), larger than the typical range for steady Antarctic hydrology ($k_c = 0.05 \text{ m}^{3/2} \text{ kg}^{-1/2}$; Dow and others, 2022) and at the upper end of the range considered by Hager and others (2022) (0.005 – $0.5 \text{ m}^{3/2} \text{ kg}^{-1/2}$). The remainder of the model parameter values in Table 1

are within the range found in the literature for GlaDS and similar models (e.g., Werder and others, 2013; Downs and others, 2018; Cook and others, 2022; Dow and others, 2022; Hager and others, 2022).

S1.2 Cavity opening mechanisms

By default, GlaDS allows cavities to creep open when water pressure exceeds ice overburden. Fundamentally, we expect that unrepresented physics would evolve the drainage system towards a state with $p_w \leq p_i$ (e.g., Tsai and Rice, 2010; Dow and others, 2015). We have also found that allowing creep opening in the model setup presented here results in unrealistic channel configurations and water pressure. When the drainage system becomes overpressurized and creep opening is allowed, cavities expand to heights much larger than the bed bump height, inhibiting channelization within overpressurized regions. With channels only developing in regions with pressure below overburden, the model struggles to evacuate meltwater from the enlarged cavities, resulting in water pressure above overburden for the entire melt season. Therefore, for conceptual and practical reasons, we have set the creep constant $\tilde{A} = 0$ where $p_w > p_i$, capping the cavity height at the bed bump height. This cap is a practical way to maintain channels within overpressurized regions and appears to be a better modelling choice than the default behaviour to overcome unrepresented physical mechanisms. While the creep constant when $p_w > p_i$ is not often reported in tables of GlaDS model parameters (e.g., Werder and others, 2013; Cook and others, 2020; Dow and others, 2022; Ehrenfeucht and others, 2023), other models cap effective pressure $N \geq 0$ to avoid cavities and channels opening by ice creep (e.g., Hoffman and others, 2018; Sommers and others, 2018).

S1.3 Distributed flux models

S1.3.1 Darcy-Weisbach turbulent flow exponents

For a fluid with density ρ , the Darcy-Weisbach equation computes the fluid potential gradient required for the fluid to flow with velocity v according to (e.g., Moody, 1944; Brown, 2002),

$$|\nabla\phi| = f_D \frac{\rho}{2} \frac{v^2}{D_H} \quad (\text{S.1})$$

for Darcy friction factor f_D and hydraulic radius D_H . For flow through an arbitrarily shaped conduit, the hydraulic radius is $D_H = 4A/P$, where A is the cross-sectional area and P is the perimeter.

The Darcy-Weisbach equation is an empirical equation, yet with roots in Bernoulli's principle, developed

52 for computing the design requirements of pipe flow (e.g., Brown, 2002). From a practical perspective, the
 53 friction factor f_D would be computed from known material properties (i.e., material roughness) and the
 54 intended Reynolds number of the flow (i.e., from the Moody diagram; Moody, 1944), in order to compute
 55 the required pipe radius, for example. The Darcy-Weisbach equation is useful for subglacial drainage
 56 modelling since it is simple compared to solving the fluid momentum balance that is applicable in both
 57 laminar and turbulent regimes and for flow through “full” drainage elements.

58 It is instructive to rewrite Eq. (S.1) for the case of flow through a pipe with a circular cross section to see
 59 how the “flow exponents” α and β in the subglacial discharge parameterization arise (Eq. 12 of Werder and
 60 others, 2013). In this case, the hydraulic radius reduces to $D_H = D$ for pipe diameter D . For a discharge
 61 Q and area S , the velocity must be $v = Q/S$. All together, and taking $D = 2\sqrt{\frac{S}{\pi}}$ (compare to Eq. 13 of
 62 Werder and others, 2013), Eq. (S.1) can be written in the form of the GlaDS flux parameterization

$$Q = \frac{2}{\sqrt{\rho f_D \pi^{1/4}}} S^{5/4} |\nabla \phi|^{1/2}. \quad (\text{S.2})$$

63 From pipe-flow experiments and from the Moody diagram (Fig. 1), fully turbulent flow corresponds to
 64 a constant f_D . This means that, in the fully turbulent limit, this form derived from the Darcy-Weisbach
 65 equation is identical to the GlaDS flux parameterization with $\alpha = 5/4$ and $\beta = 3/2$ (Werder and others,
 66 2013).

67 Similar steps lead to the conclusion that $\alpha = 3/2$ for the distributed flow parameterization. In the
 68 spirit of the bulk-averaged approach to the drainage through a “subglacial water sheet” (e.g., Werder and
 69 others, 2013), consider flow at velocity v through a gap with average height h . Before we proceed with Eq.
 70 (S.1), we need an expression for the hydraulic radius. For parallel lower and upper boundaries (i.e., the
 71 gap height is exactly h everywhere), the hydraulic for flow through a gap of width w is

$$D_H = 4 \frac{wh}{2(w+h)} \approx 2h, \quad (\text{S.3})$$

72 where the approximation assumes that $w \gg h$. This approximation should hold in the subglacial
 73 context, where the width of the flow is on the scale of kilometers and the vertical extent is on the scale
 74 of meters. While we don’t envision the gap height being uniformly h everywhere for physical drainage
 75 elements comprising the “distributed drainage system”, the shallow aspect ratio of the subglacial drainage
 76 system should ensure the applicability of this hydraulic radius. By writing the discharge-per-unit-width as

77 $q = vh$, we can write Eq. (S.1) in the context of sheet-like flow as

$$q = 2 \frac{1}{\sqrt{\rho f_D}} h^{3/2} |\nabla \phi|^{1/2}. \quad (\text{S.4})$$

78 Once again, taking the fully turbulent limit and assuming f_D is constant (Fig. 1), this expression is in
 79 the form of the GlaDS sheet-flow model with $\alpha = 3/2$ (rather than $5/4$, as is usually suggested for fully
 80 turbulent flow; Werder and others, 2013) and $\beta = 3/2$.

81 *S1.3.2 Implementation of the transition flux parameterization*

82 The Transition flux parameterization as presented in the main text is

$$-k_s h^3 \nabla \phi = \mathbf{q} + \omega \text{Re} \left(\frac{h}{h_b} \right)^{3-2\alpha_s} \mathbf{q}, \quad (\text{S.5})$$

83 for conductivity k_s , hydraulic potential ϕ , discharge-per-unit-width q , Reynolds number Re , sheet
 84 thickness h , bed bump height h_b , and turbulent flow exponent α_s . Here we present the brief steps required
 85 to explicitly solve the parameterization for the flux q . Substituting the Reynolds number ($\text{Re} = \frac{q}{\nu}$) yields
 86 a quadratic equation in terms of $q = |\mathbf{q}|$,

$$-k_s h^3 \nabla \phi = \mathbf{q} + \frac{\omega}{\nu} \left(\frac{h}{h_b} \right)^{3-2\alpha_s} q \mathbf{q}. \quad (\text{S.6})$$

87 This equation can be solved exactly for \mathbf{q} ,

$$\mathbf{q} = -\frac{\nu}{2\omega} \left(\frac{h_0}{h} \right)^{3-2\alpha_s} \left(-1 + \sqrt{1 + 4 \frac{\omega}{\nu} \left(\frac{h}{h_0} \right)^{3-2\alpha_s} k h^3 |\nabla \phi|} \right) \frac{\nabla \phi}{|\nabla \phi|}. \quad (\text{S.7})$$

88 This expression simplifies for the $\alpha_s = 3/2$ case (i.e., for the Turbulent 3/2 model),

$$\mathbf{q} = -\frac{\nu}{2\omega} \left(-1 + \sqrt{1 + 4 \frac{\omega}{\nu} k h^3 |\nabla \phi|} \right) \frac{\nabla \phi}{|\nabla \phi|}. \quad (\text{S.8})$$

89 This simplified expression and Eq. (S.7) are no more difficult to evaluate numerically than the standard
 90 GlaDS parameterization, as they both require evaluating a single spatial derivative ($\nabla \phi$), and are written
 91 explicitly so no iteration is required.

92 *S1.3.3 Turbulent conductivity scaling*

93 Since the conductivity for the turbulent model is specified in different units (that depend on the turbulent
94 flow exponent α_s) than the laminar and transition models, the turbulent conductivity needs to be scaled
95 appropriately to ensure a valid comparison between flux models.

96 The critical water layer thickness that results in a Reynolds number equal to the transition threshold
97 is computed from the laminar model by setting $\omega\text{Re} = \omega q/\nu = 1$,

$$h_{\text{crit}}^3 = \frac{\nu}{\omega} \frac{1}{k_s |\nabla\phi|}. \quad (\text{S.9})$$

98 We solve for the turbulent conductivity by setting the laminar and turbulent models equal with $h = h_{\text{crit}}$,

$$k_t h_{\text{crit}}^{\alpha_s} |\nabla\phi|^{1/2} = k_s h_{\text{crit}}^3 |\nabla\phi|. \quad (\text{S.10})$$

99 Therefore, the turbulent conductivity is given by

$$k_t = k_s h_{\text{crit}}^{3-\alpha_s} |\nabla\phi|^{1/2}. \quad (\text{S.11})$$

100 When $\alpha_s = 3/2$, the $\nabla\phi$ -dependence drops out of the expression for k_t . When $\alpha_s = 5/4$, the potential
101 gradient is set to the average value assuming water pressure is equal to overburden for a given geometry.

102 **S1.4 Surface melt forcing**

103 To derive realistic yet idealized surface melt forcing, we use temperatures recorded at the PROMICE
104 KAN_L weather station (How and others, 2022) to drive a degree-day melt model. Before using the
105 temperature record to drive the melt model, we find a melt year that is representative according to the
106 total volume and duration of surface melt.

107 For the record of diurnal mean temperatures spanning 2009–2022, we compute the cumulative positive
108 degree-days as a proxy for total melt volume and the melt season duration. Cumulative positive degree-
109 days are computed directly from the diurnal mean temperatures. The beginning of the melt season is
110 defined as the first day with positive diurnal mean temperatures on the n following consecutive days. The
111 end of the melt season is defined analogously as the last day with positive diurnal mean temperatures on
112 the n preceding consecutive days. We compute the melt season duration for $n = 2, 3, 5$ days to derive

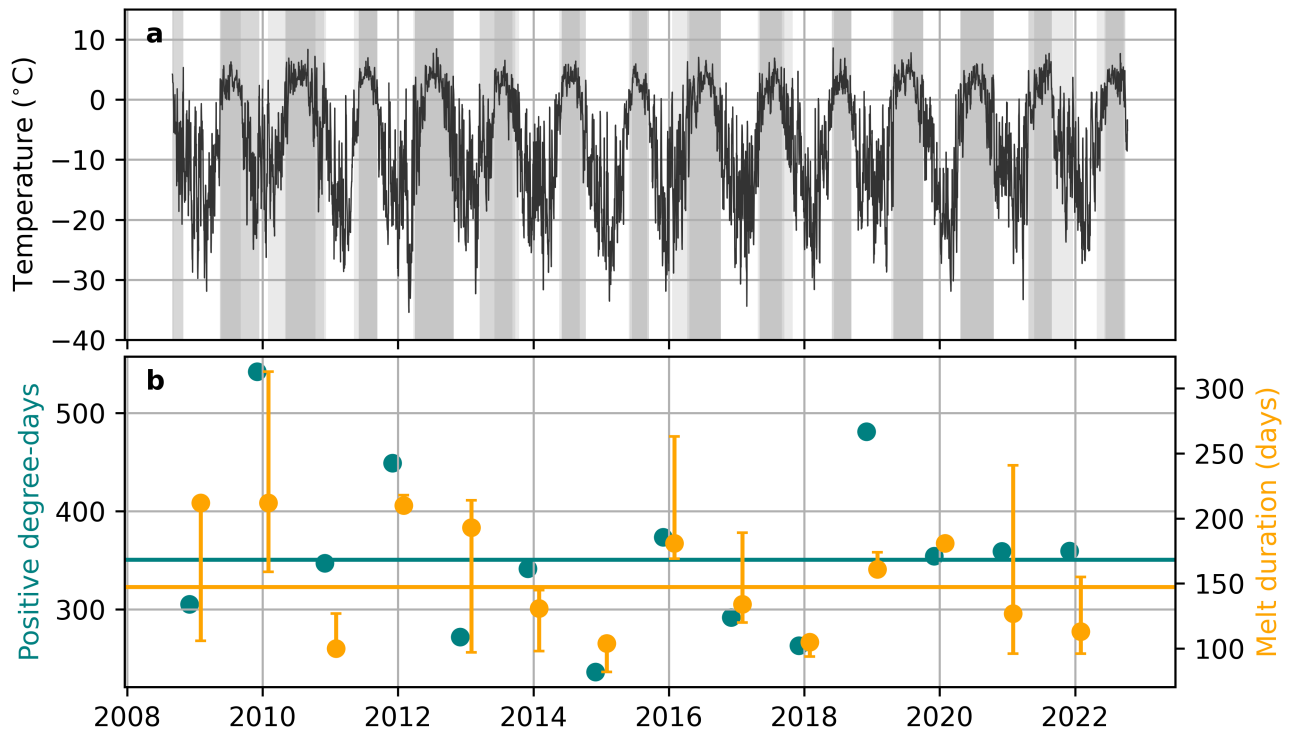


Fig. S1. KAN_L automated weather station (AWS) temperature record (How and others, 2022), positive degree-days and melt season durations. (a) Diurnal mean AWS temperature. Shaded grey regions indicate the melt season, defined as consecutive positive-degree days for 2, 3, or 5 days to derive a range of melt season durations. (b) Cumulative positive degree-days (left axis) and melt season duration (right axis). Horizontal lines indicate the median positive degree-days and melt season duration. Error bars for the melt season duration show the minimum and maximum melt season duration from the ensemble of melt season definitions. The 3-day definition is used to plot the representative melt season duration. Based on the total positive degree-days and melt season duration near the 2009–2022 medians, we select 2014 as the representative melt year.

113 lower and upper bounds on the melt season duration. The difference between the bounds is used as an
 114 approximate indicator of short-duration melt events outside the core melt season with nearly continuous
 115 positive temperatures.

116 The 14-year record of diurnal mean temperature record, positive degree-days and melt season duration
 117 are compared in Fig. S1. Based on the median positive degree-days and melt season duration, we choose
 118 2014 as a representative melt year.

119 S1.5 Moulin design

120 Surface runoff is integrated within supraglacial catchments and routed through moulins to localize melt
 121 inputs to the bed. For the synthetic domain geometry, catchment and moulin designs are generated from a
 122 space-filling Latin hypercube design that is scaled to approximate the observed relative density of moulins

123 within elevation bands.

124 The observed density of moulins as a function of elevation is derived from satellite-based supraglacial
125 hydrology mapping of a portion of the southwest Greenland ice sheet (Yang and Smith, 2016) (Fig. S2).
126 Moulin designs are generated by drawing a sample of y (across-glacier) and z (elevation) coordinates,
127 which are then scaled to fill the complete domain-width and elevation range, with the elevation samples
128 conforming to the observed distribution. The inverted surface elevation function is used to transform the
129 elevations to x positions. The (x, y) coordinates are then used as the centers of supraglacial catchments
130 and moulins are placed at the lowest node within each catchment subject to:

- 131 1. $x \geq 5$ km to mimic the effects of high surface slopes and crevasses (Werder and others, 2013),
- 132 2. the minimum distance between moulins is 2.5 km, and
- 133 3. moulins aren't placed on boundary nodes.

134 Surface melt is integrated within each catchment and instantaneously routed through the catchment into
135 the outlet moulin.

136 Fig. S2 illustrates the process to obtain a moulin design. The deviation between the synthetic and
137 Yang and Smith (2016) density between 500 and 600 m elevation is due to the first condition above. This
138 condition results in four moulins placed at $x = 5$ km, and since the area within the 500 to 600 m elevation
139 bin is small relative to the area within higher bins due to the high surface slope near the terminus, the
140 observed density is artificially high between 500 and 600 m. The remainder of the deviations are a result
141 of the random nature of the moulin placement scheme.

142 **S1.6 Mesh sensitivity**

143 To assess the sensitivity of modelled subglacial drainage to the numerical mesh, we ran steady state
144 simulations with 10 meshes with the number of nodes varying between 107 and 16169 (Fig. S3). Each mesh
145 refinement case is run with identical geometry and parameters. The total prescribed moulin discharge is
146 identical, but the position of moulins changes since moulins must be placed on mesh nodes. To reposition
147 moulins for each mesh, moulin positions are interpolated from the reference case (4156 nodes) to the
148 nearest node in the mesh under consideration, and the moulin discharge is directly transferred to the
149 newly repositioned moulin. For the coarse meshes, this approach means that sometimes multiple moulins

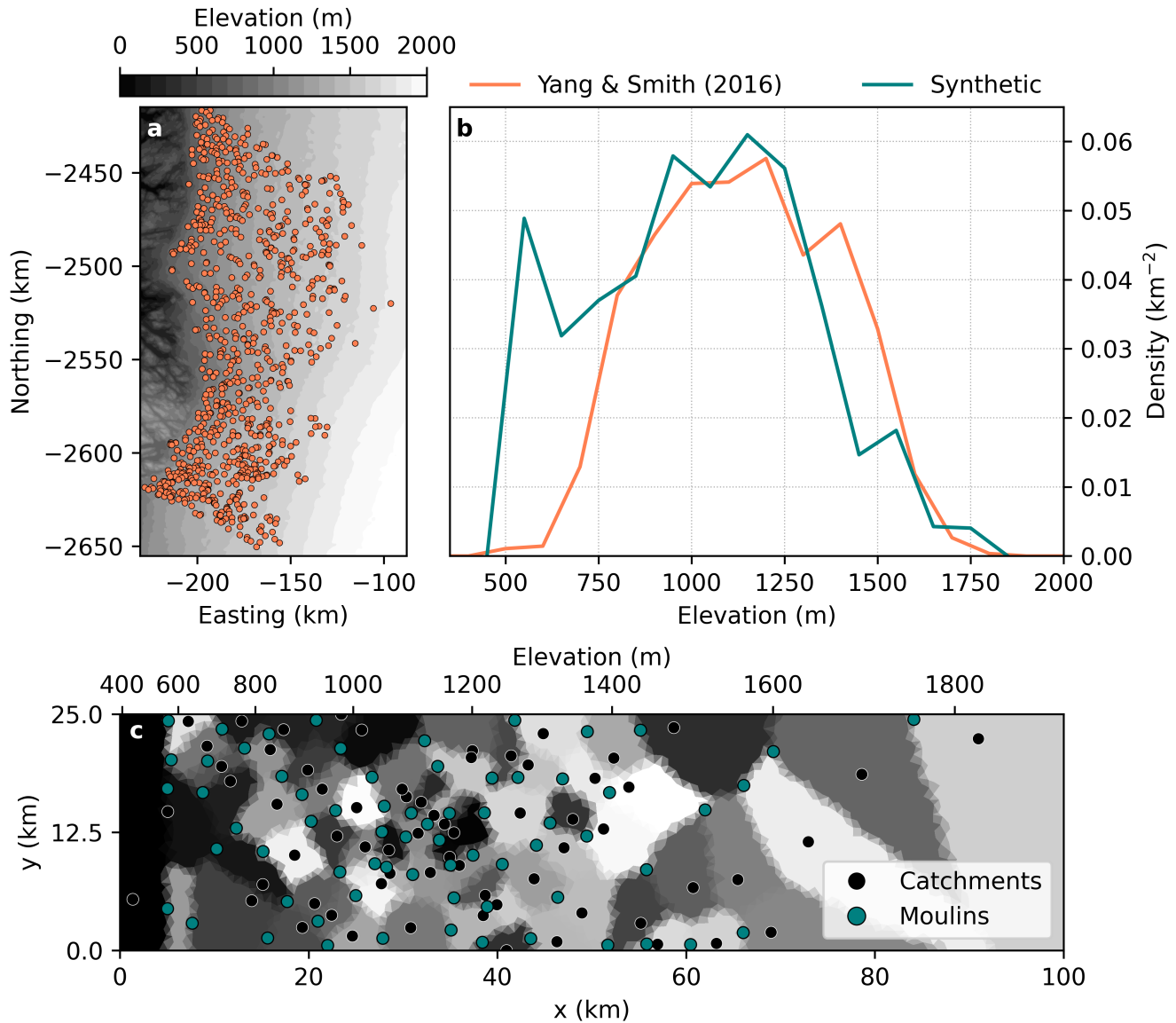


Fig. S2. Summary of the scheme used to generate the synthetic moulin design. (a) ArcticDEM 500 m mosaic surface elevation (Porter and others, 2018) and moulin positions mapped from a 19 August 2013 panchromatic Landsat 8 image by Yang and Smith (2016). (b) Moulin density in 100 m elevation bins derived from the Yang and Smith (2016) dataset and for the synthetic moulin design. (c) Synthetic catchment centers, moulin positions, and catchment areas (coloured polygons), with distance (bottom) and surface elevation (top) scales.

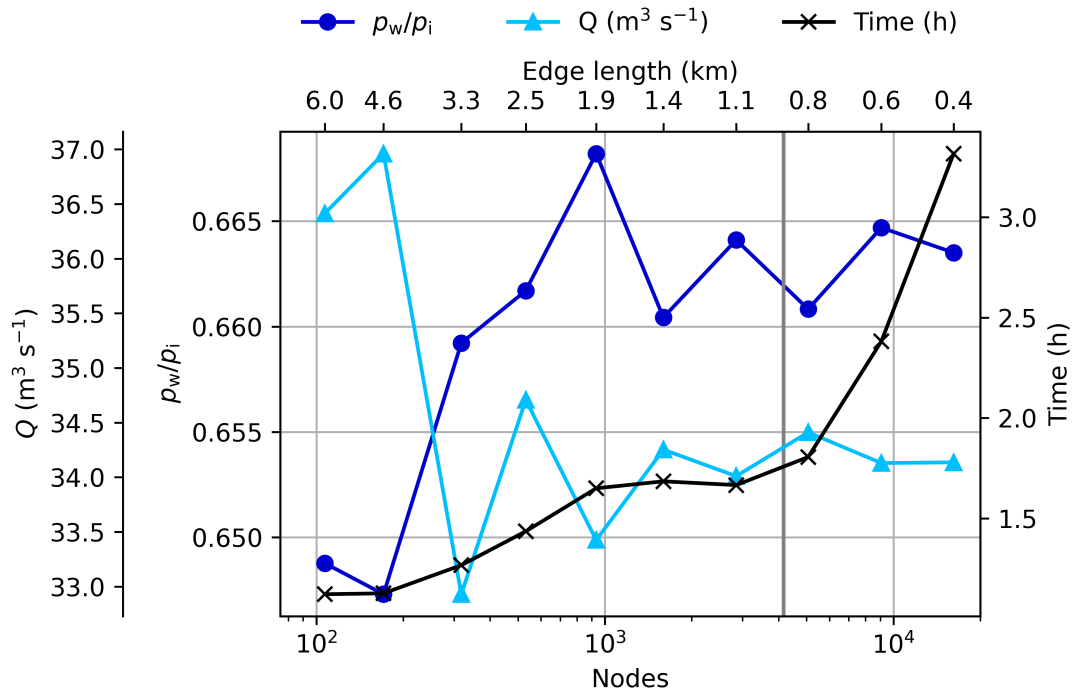


Fig. S3. Results of the mesh refinement test. Mean floatation fraction (inner left axis), total channel discharge (outer left axis) at $x = 30 \pm 2.5$ km and total model runtime (right axis) for 10 meshes with increasing number of nodes. The default mesh in the main text contains 4156 nodes (indicated by the vertical grey line).

150 from the reference mesh are placed at the same node. When this happens, the moulin discharge is the sum
 151 of the discharge for original moulins that are now co-located.

152 The floatation fraction and channel discharge converge as the number of nodes increases as well as
 153 expected for GlaDS (Fig. S3), where the random orientation of edges has some influence on channel
 154 positions (e.g., Werder and others, 2013, Fig. A1). Some of the residual variance in the mesh refinement
 155 curves may also be a result of interpolating moulins to new node positions. Nevertheless, the reference
 156 mesh (4156 nodes) represents an appropriate balance between convergence and model runtime.

157 **S2 RESULTS**

158 **S2.1 Steady state comparison**

159 Fig. S4 shows the floatation fraction, Reynolds number, and channel discharge at the end of the 100 year
160 spin-up, when the drainage system has reached a steady state.

161 **S2.2 Tabulated floatation fraction and computation time**

162 Table S1 compares winter and summer pressures for all models within 5 km-wide bands centred at $x = 15$,
163 30, and 70 km. Table S2 compares the total computation time for the main series of simulations. These
164 times represent the total time for spinup and transient simulations.

165 **S2.3 KAN scenario Reynolds number**

166 To assess the consistency of the laminar and turbulent assumptions for the KAN forcing scenario, Fig. S5
167 presents the seasonal evolution of the Reynolds number. All models result in turbulent flow in the early
168 melt season, with the highest Reynolds number produced by the laminar model in the early melt season.

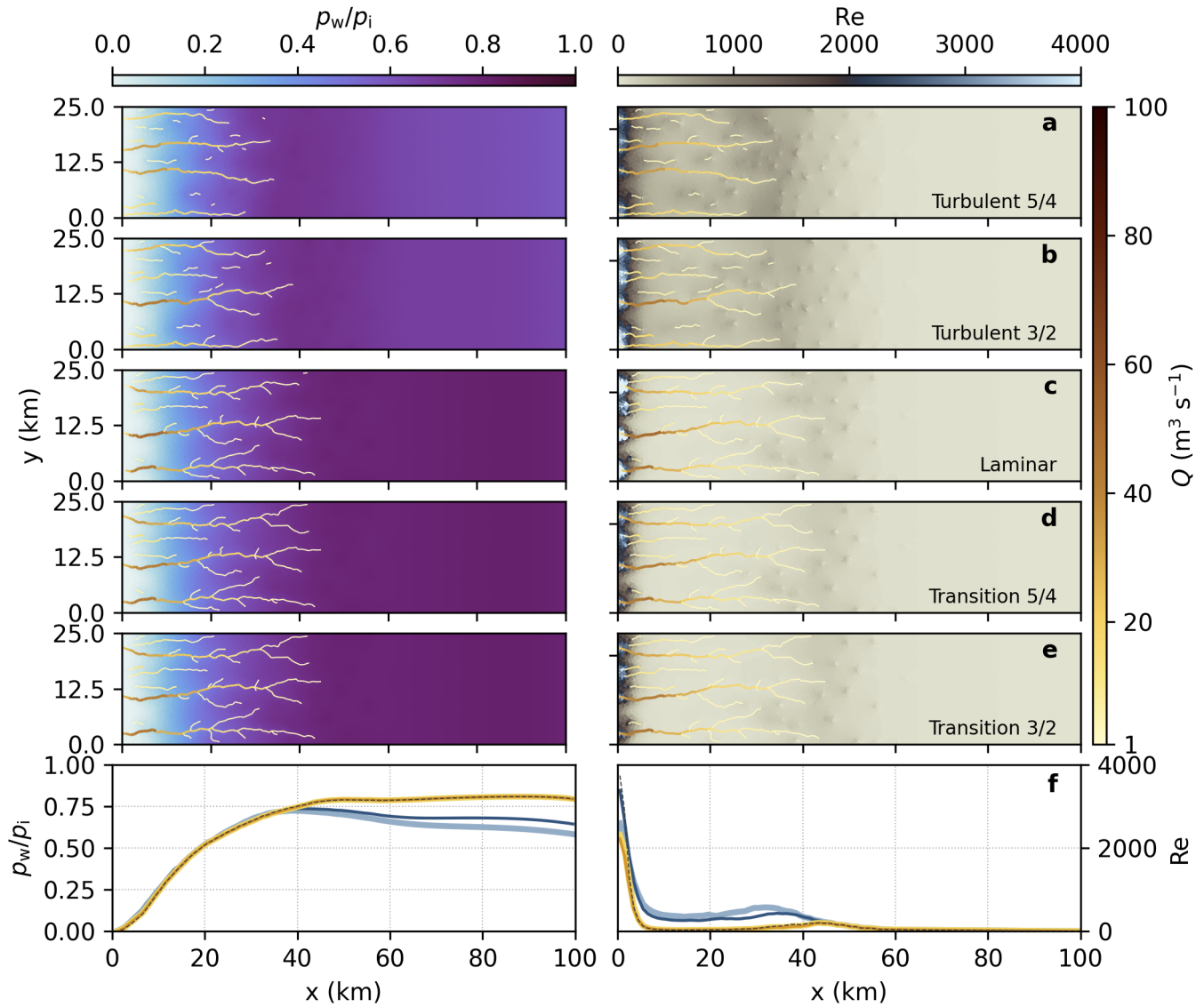


Fig. S4. Flootation fraction and Reynolds number for steady state simulation. (a–e) Flootation fraction (left) and Reynolds number ($Re = \frac{q}{\nu}$; right) for the five sheet-flux parameterizations. (f) Width-averaged profiles of floatation fraction (left) and Reynolds number (right).

Table S1. Water pressure normalized by overburden (floatation fraction) for synthetic and KAN surface melt forcing scenarios. Winter floatation fraction is computed as the width-averaged value at the specified position (± 2.5 km) during the two months preceding the initial onset of surface melt. The reported summer floatation fraction values are the 95th-percentile width-averaged water pressure produced during the melt season within ± 2.5 km of the specified position. The bracketed number beside summer floatation fractions for the KAN scenario indicates the number of days water pressure exceeded overburden. Water pressure does not exceed overburden in the Synthetic scenario. “Turb” refers to turbulent models, “Trans” refers to Transition models, and the fraction (5/4 or 3/2) identifies the turbulent flow exponent α_s value.

Scenario	Season	Distance from ter- minus	Floatation fraction				
			Turb 5/4	Turb 3/2	Laminar	Trans 5/4	Trans 3/2
Synthetic	Winter	15 km	0.138	0.231	0.466	0.466	0.468
		30 km	0.430	0.511	0.670	0.671	0.672
		70 km	0.616	0.675	0.798	0.798	0.798
	Summer	15 km	0.751	0.742	0.698	0.728	0.757
		30 km	0.829	0.826	0.811	0.828	0.847
		70 km	0.669	0.696	0.799	0.799	0.800
KAN	Winter	15 km	0.144	0.234	0.466	0.467	0.468
		30 km	0.436	0.513	0.670	0.671	0.672
		70 km	0.620	0.676	0.798	0.798	0.798
	Summer	15 km	0.98 (8)	1.03 (9)	0.960 (6)	1.03 (10)	1.07 (10)
		30 km	1.04 (9)	1.02 (10)	0.997 (6)	1.01 (9)	1.01 (11)
		70 km	0.771 (0) (0)	0.785 (0)	0.849 (0)	0.851 (0)	0.857 (0)

Table S2. Total simulation runtime in hours for synthetic and KAN scenarios (spinup and transient) and for each flux parameterization. Simulations were carried out in serial on Intel E5-2683 CPUs with 4G of memory on the Compute Canada Cedar cluster.

Model	Synthetic scenario	KAN scenario
Turbulent 5/4	0.78	8.37
Turbulent 3/2	0.64	10.39
Laminar	0.57	8.52
Transition 5/4	0.63	9.51
Transition 3/2	0.69	12.60

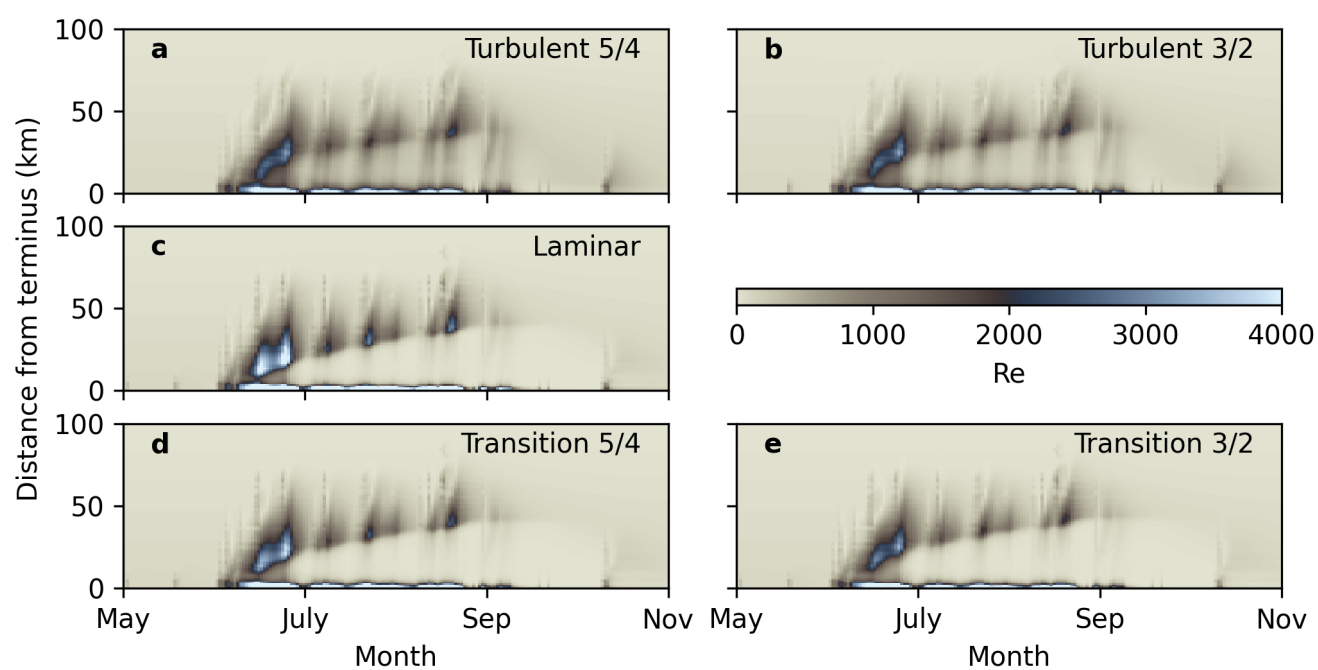


Fig. S5. Seasonal evolution of width-averaged Reynolds number with KAN forcing for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d), and Transition 3/2 (e) models.

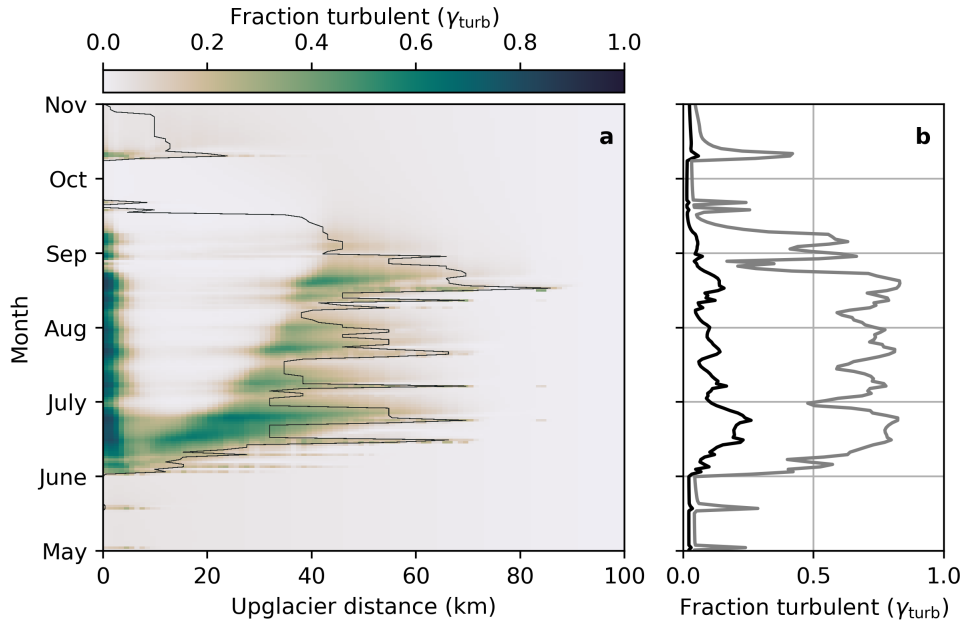


Fig. S6. Seasonal transitions between laminar and turbulent flow for the Transition 3/2 model with KAN forcing. (a) Width-averaged fraction of turbulent flow (γ_{turb} ; Eq. S.13). The thin superimposed black line delineates the maximum upglacier extent of channels with discharge $Q > 1 \text{ m}^3 \text{ s}^{-1}$. (b) Domain-averaged (black) and maximum width-averaged (grey) fraction turbulent.

169 S2.4 Seasonal laminar turbulent partitioning

170 The partitioning between laminar and turbulent flow in the transition model can be quantified by comparing
 171 the size of the terms on the right-hand-side of Eq. (S.5). The contribution of laminar and turbulent flows
 172 are therefore

$$\gamma_{\text{laminar}} = \frac{q}{kh^3|\nabla\phi|} \quad (\text{S.12})$$

$$\gamma_{\text{turb}} = \frac{\frac{\omega}{\nu} \left(\frac{h}{h_b}\right)^{3-2\alpha_s} q^2}{kh^3|\nabla\phi|}. \quad (\text{S.13})$$

173 Figure S6 shows the turbulent fraction, γ_{turb} , for the KAN forcing scenario. Flow is laminar outside
 174 the melt season and above the maximum melt extent. The model also predicts laminar sheet-flow within
 175 the region of the well-developed channel network during the melt season (i.e., between ~ 5 and ~ 30 km
 176 from July to September). Flow partially transitions to turbulent within the upper reaches of the channel
 177 network, immediately downstream of the $Q > 1 \text{ m}^3 \text{ s}^{-1}$ contour. Flow is turbulent within the lowest ~ 5
 178 km during the melt season, however this is attributed to boundary effects.

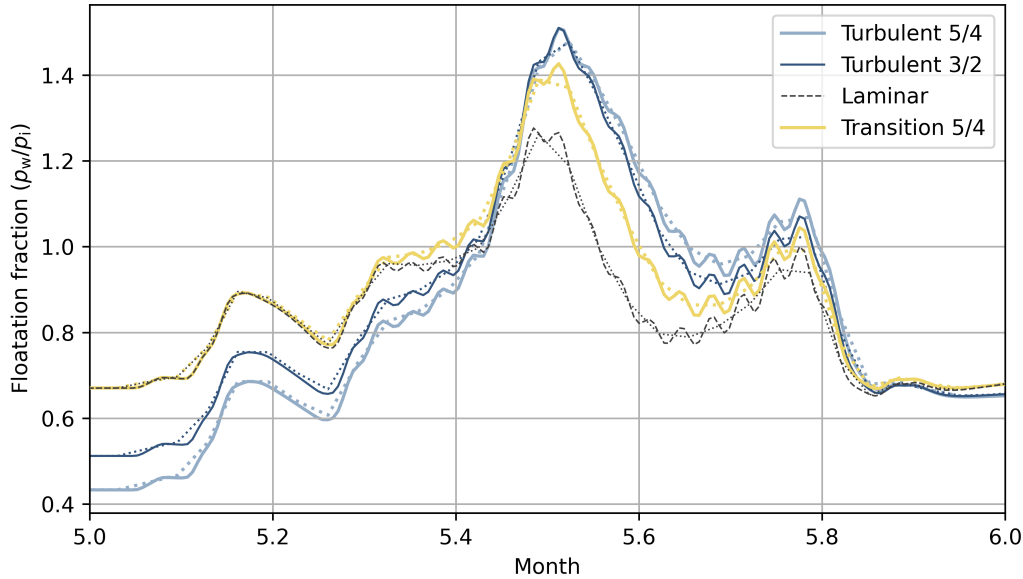


Fig. S7. Average floatation fraction within $x = 30 \pm 2.5$ km with (solid) and without (dotted) diurnal variations in surface melt forcing for the first month of the KAN scenario melt season.

179 S2.5 Diurnal melt-rate variations

180 To assess the impact of diurnal melt-rate variations on modelled water pressure, diurnals are added to
 181 the prescribed surface melt rate. The diurnal surface melt rate, \dot{m}_d , is computed from the seasonal KAN
 182 scenario melt rate \dot{m} (Eq. 6) following

$$\dot{m}_d(z, t; \Gamma) = \dot{m}(z, t; \Gamma) \left(1 - 0.5 \cos \left(\frac{2\pi t}{T_{\text{day}}} \right) \right), \quad (\text{S.14})$$

183 for t the time in seconds and T_{day} the number of seconds per day.

184 Fig. S7 compares modelled floatation fraction within $x = 30 \pm 2.5$ km with and without diurnal variations
 185 in surface melt forcing for four of the five flux parameterizations. There is little difference in floatation with
 186 and without diurnal variations, and so the relative performance of each model is not sensitive to diurnal
 187 forcing.

188 S2.6 Model verification

189 To verify the implementation of the transition models, and more broadly to compare two of the presently
 190 available implementations of GlaDS, we compare the Matlab implementation (i.e., the code from Werder
 191 and others, 2013) with the relatively newer implementation within the Ice-sheet and Sea-level System Model

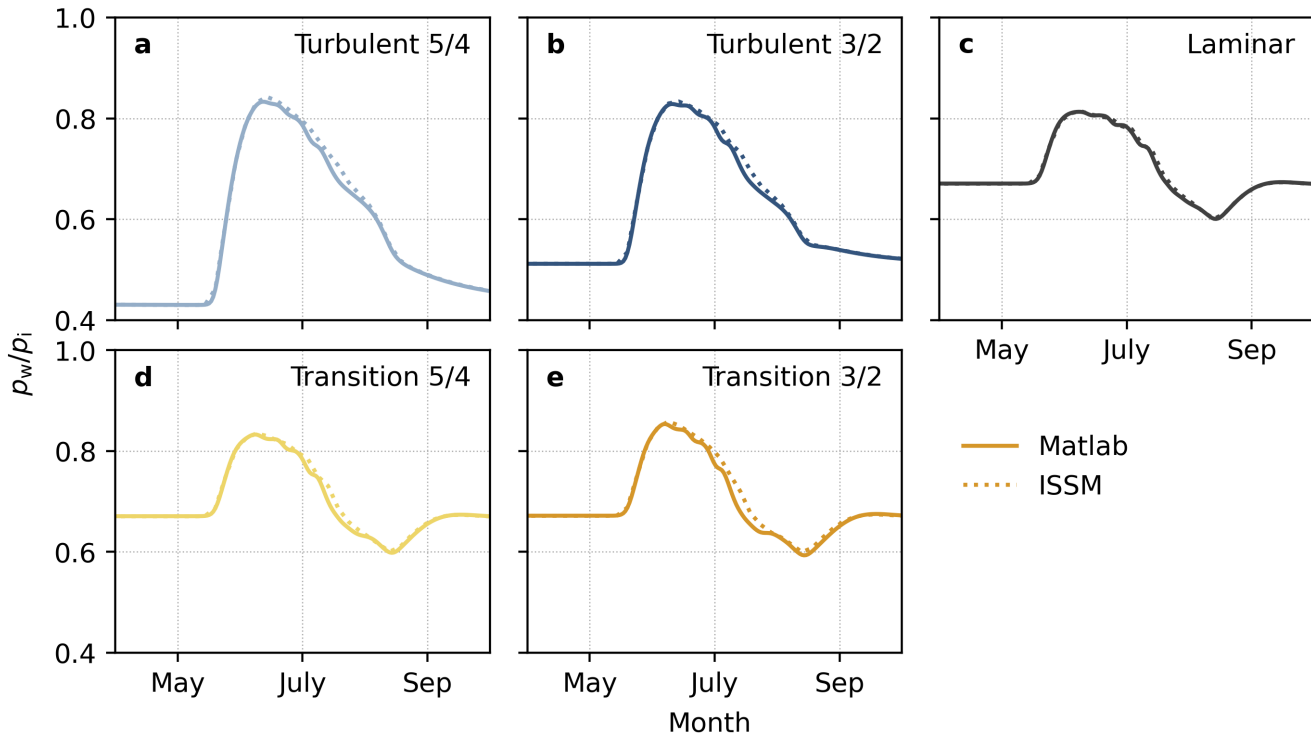


Fig. S8. Model verification against the ISSM implementation of GlaDS. Width-averaged floatation fraction 30 km from the terminus with synthetic melt forcing produced with the Matlab (solid) and ISSM (dotted) models.

192 (ISSM; Larour and others, 2012) (using branch trunk-jpl, revision 27936). We run GlaDS within ISSM
 193 using a setup identical to that described in Section 2. The water pressure produced by the two models
 194 closely agrees, with only minor differences in the behaviour following peak water pressure (Fig. S8). With
 195 these results, we are confident in the implementation of the transition models within these two independent
 196 models.

197 S3 SENSITIVITIES

198 S3.1 Melt rates

199 S3.1.1 Surface melt volume

200 The sensitivity to total surface melt volume is assessed by running the synthetic scenario with melt forcing
 201 identical to case D3 from the SHMIP experiment (Fig. S9), and by adjusting the KAN forcing to match
 202 the SHMIP D3 total melt volume (Fig. S10).

203 The KAN melt forcing is adjusted by setting the temperature lapse-rate equal to $\Gamma = -0.0075^\circ\text{C m}^{-1}$
 204 (to match SHMIP case D3) and solving for the multiplicative factor that yields identical total surface melt

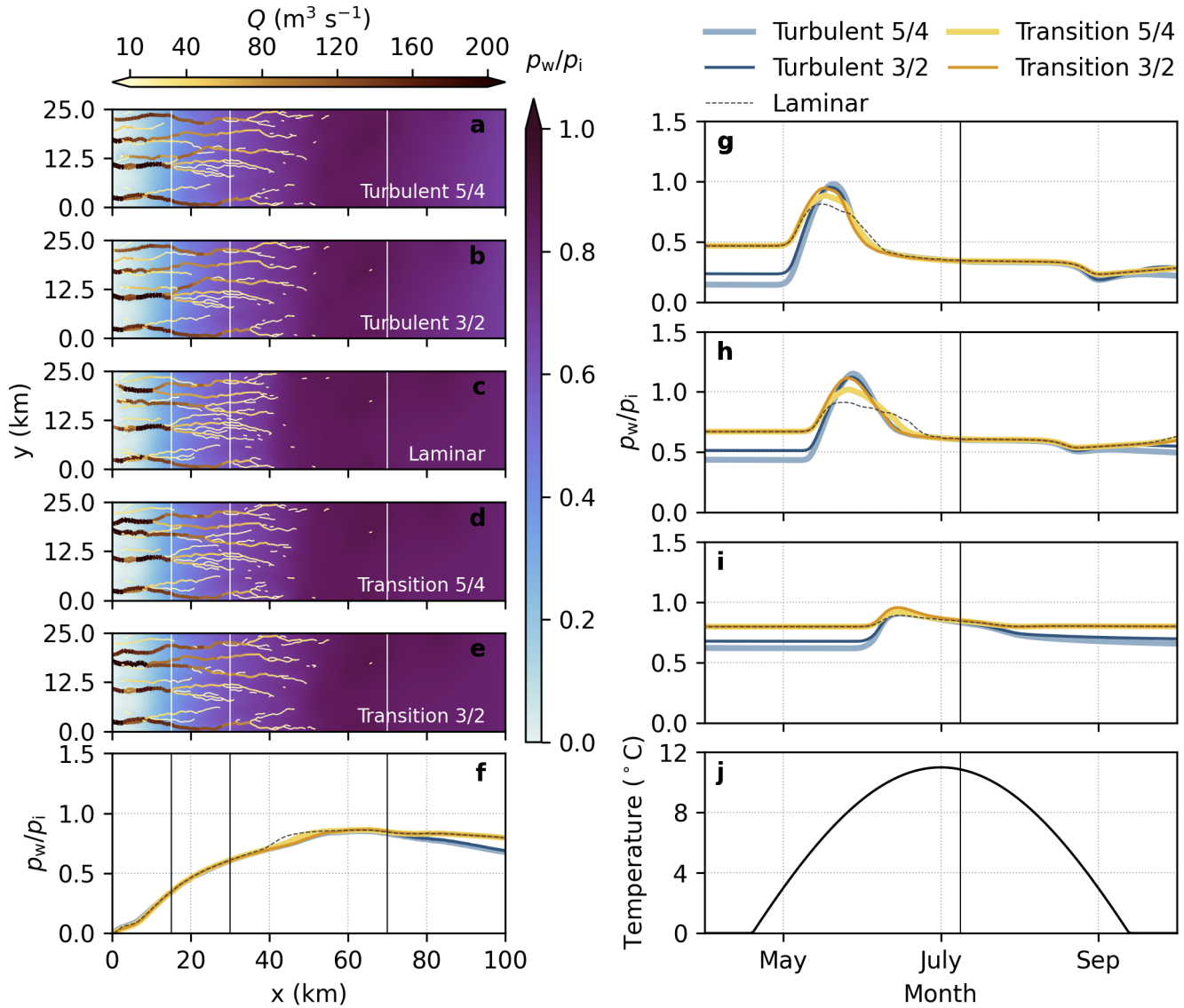


Fig. S9. Transient simulation with surface melt rates identical to SHMIP case D3. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

Table S3. Parameter adjustments for the sensitivity test.

Parameter	Default value	“High” channelization scenario	“Low” channelization scenario
k_s	0.05 Pa s ⁻¹	0.02 Pa s ⁻¹	0.1 Pa s ⁻¹
k_c	0.5 m ^{3/2} s ⁻¹	1.0 m ^{3/2} s ⁻¹	0.2 m ^{3/2} s ⁻¹
l_c	10 m	25 m	10 m

205 volumes.

206 *S3.1.2 Basal melt rate*

207 Sensitivity to basal melt rate is tested by decreasing the prescribed basal melt rate from 0.05 m w.e. a⁻¹ to
 208 0.01 m w.e. a⁻¹ (i.e., a low basal friction regime). Figure S11 compares average floatation fraction 30 km
 209 from the terminus for low and high basal melt rates with KAN surface melt forcing.

210 Decreasing the basal melt rate decreases winter water pressure for all flux parameterizations, but has
 211 a limited impact once surface melt drives drainage development (Fig. S11). The decreased winter water
 212 pressure highlights that including a realistic basal melt rate is important for obtaining broadly more realistic
 213 results and for producing a stronger late-summer pressure minimum for the laminar and transition models.

214 **S3.2 Parameter sensitivity**

215 *S3.2.1 Channelization*

216 We assess how the relative performance of the flux parameterizations changes for end-members representing
 217 minimal and extensive channelization. Table S3 lists the parameters tuned to run these scenarios. Figures
 218 S12 and S13 present the floatation fraction, channel discharge, and Reynolds number for the high channel-
 219 ization scenario. Figures S14 and S15 present the same quantities for the low channelization scenario.

220 *S3.2.2 Englacial storage*

221 The magnitude of pressure variations with KAN forcing is sensitive to the chosen englacial storage pa-
 222 rameter. To constrain this sensitivity, we run the KAN experiment with the englacial storage parameter
 223 reduced from $e_v = 1 \times 10^{-4}$ to $e_v = 2 \times 10^{-5}$. This reduction in storage shortens the timescale associated
 224 with pressure adjustments and increases the magnitude of pressure variations (Fig. S16).

225 Reducing englacial storage increases the magnitude of the spring pressure maximum (>200% of over-

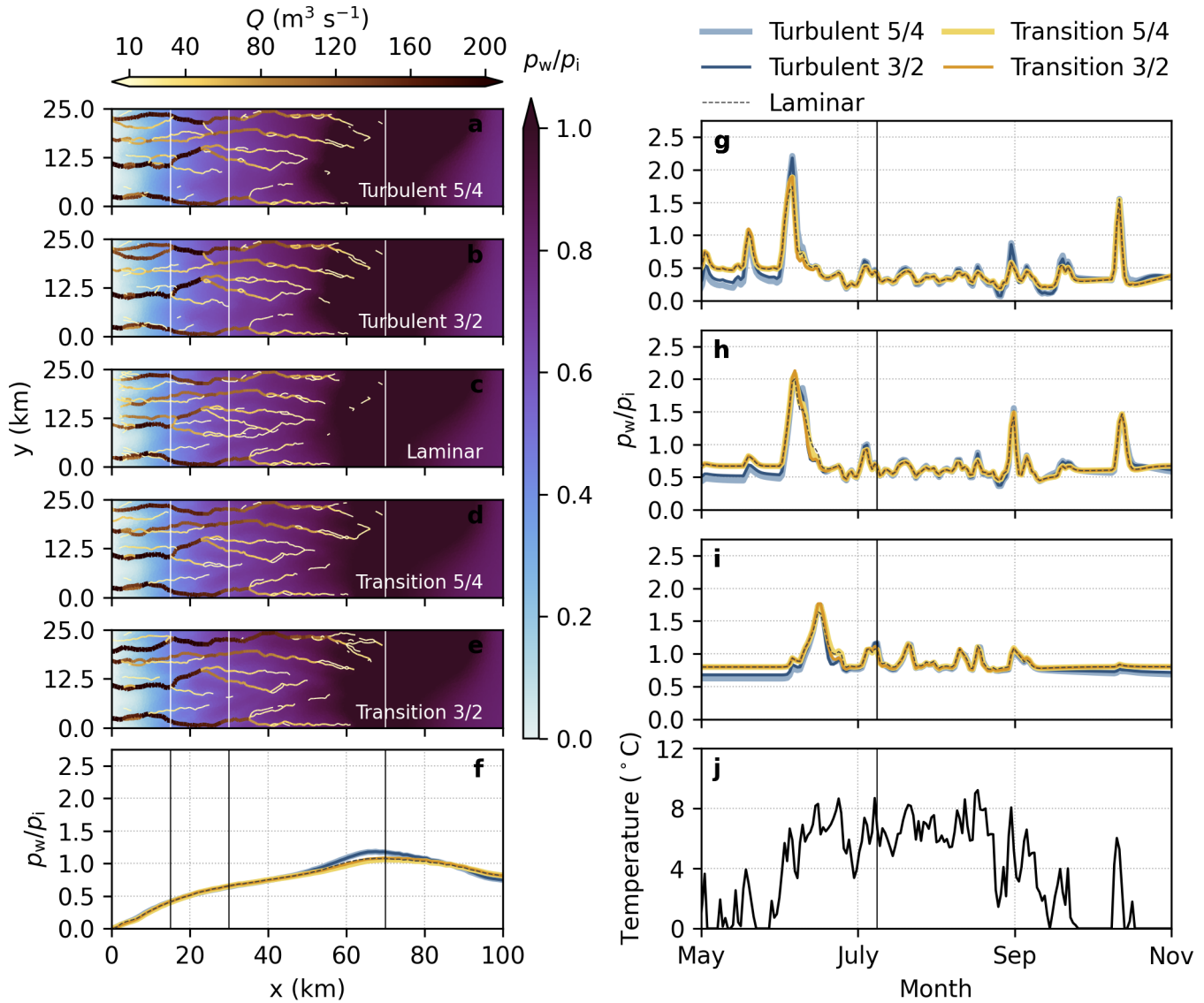


Fig. S10. Transient simulation with KAN surface melt forcing adjusted to have identical total surface melt volume as SHMIP case D3. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

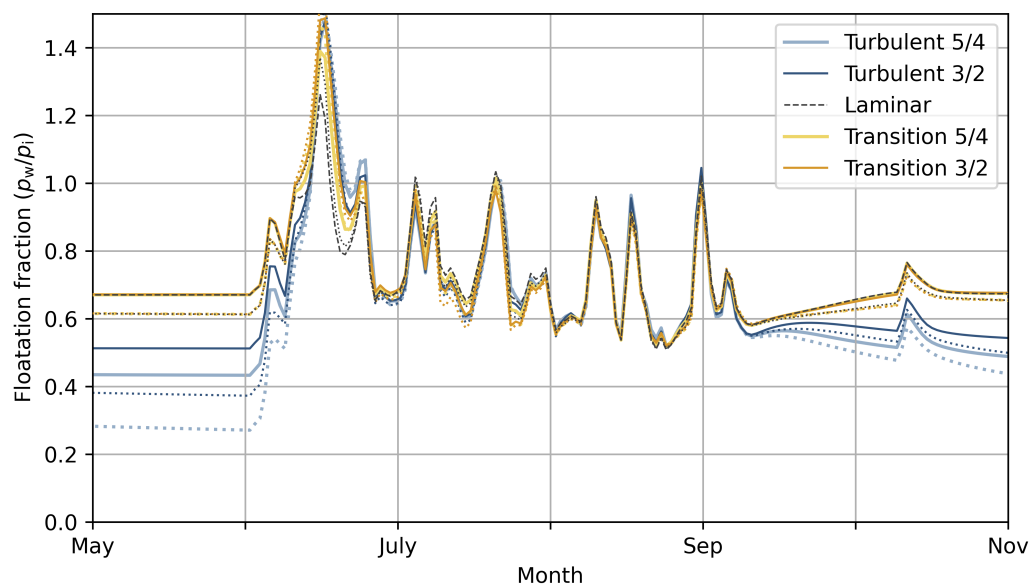


Fig. S11. Average floatation fraction within $x = 30 \pm 2.5$ km with basal melt rates of $0.01 \text{ m w.e. a}^{-1}$ (dotted) and $0.05 \text{ m w.e. a}^{-1}$ (solid; dashed for the laminar model for clarity) with KAN surface melt forcing.

226 burden for the turbulent models), but this does not change the ordering of models in terms of winter water
 227 pressure, late-summer pressure minima, the spring pressure maximum, or internal consistency.

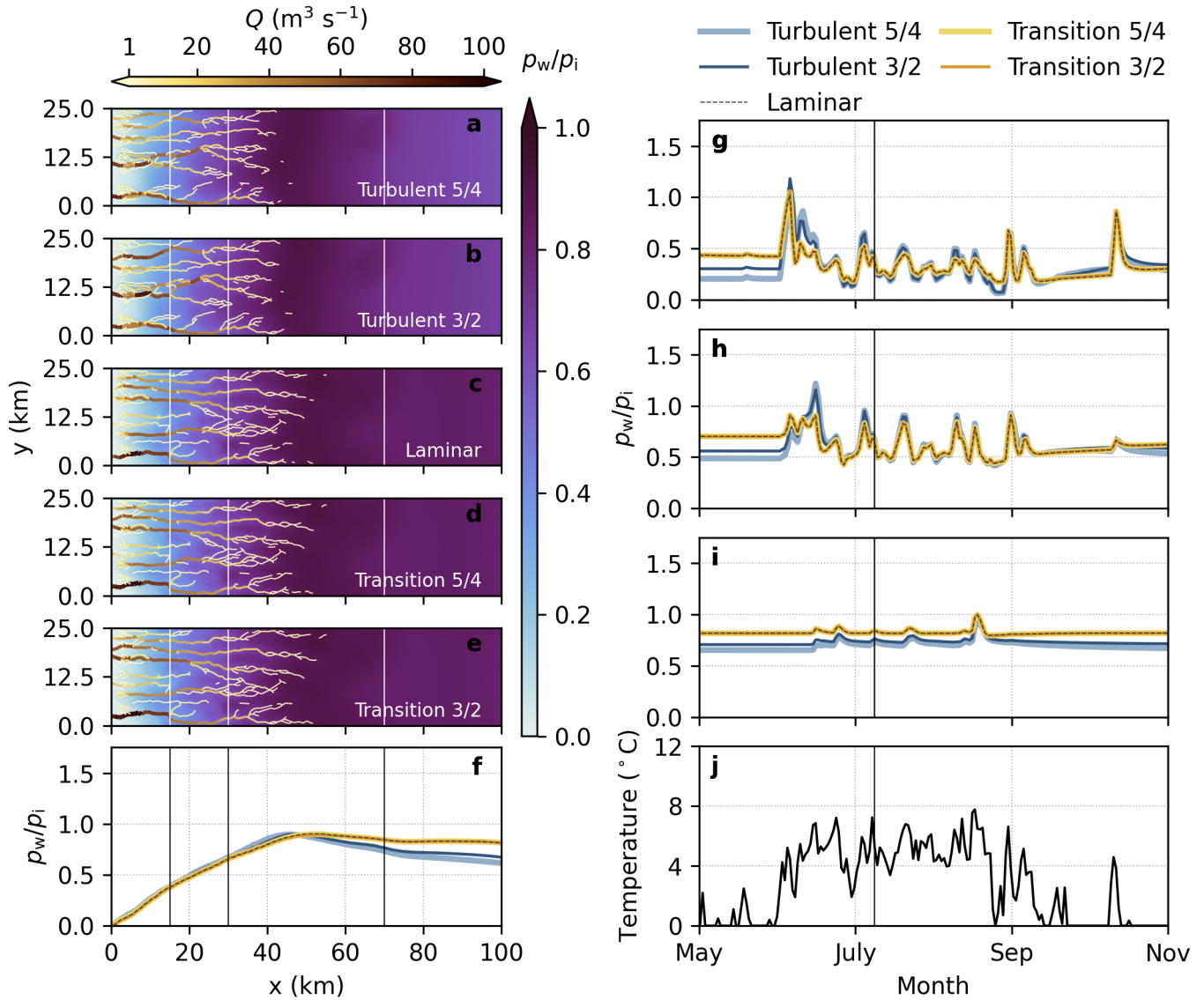


Fig. S12. Drainage configuration for “high” scenario parameters as in Table S3. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

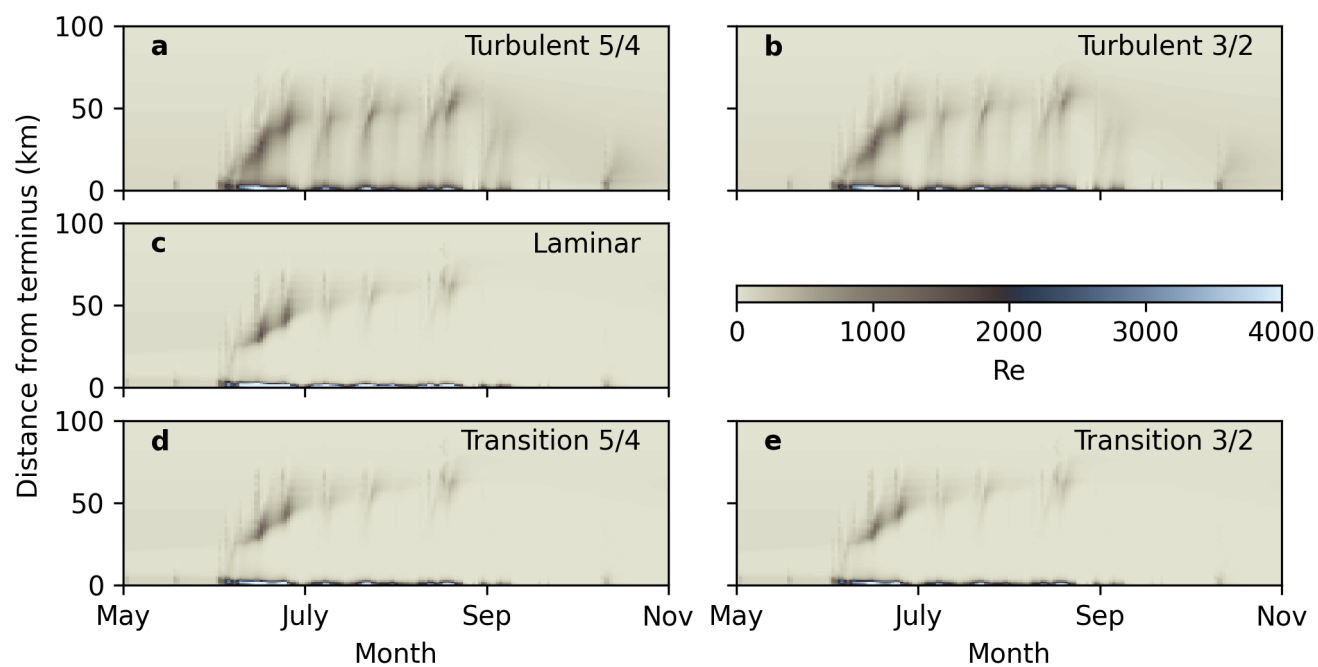


Fig. S13. Seasonal evolution of width-averaged Reynolds number and channel discharge for “high” scenario parameters as in Table S3 for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d), and Transition 3/2 (e) models.

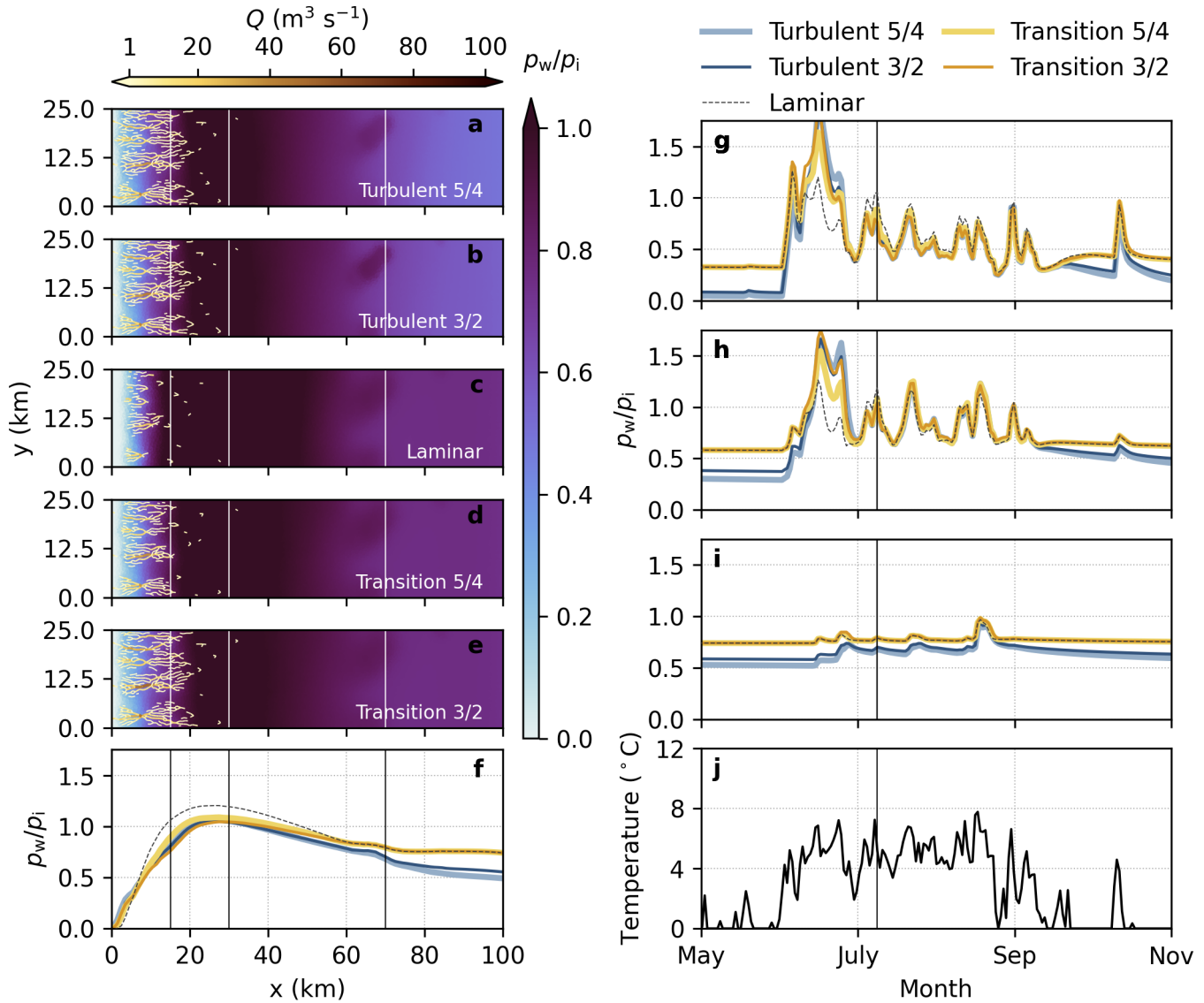


Fig. S14. Drainage configuration for “low” scenario parameters as in Table S3. Floataction fraction $\frac{z_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

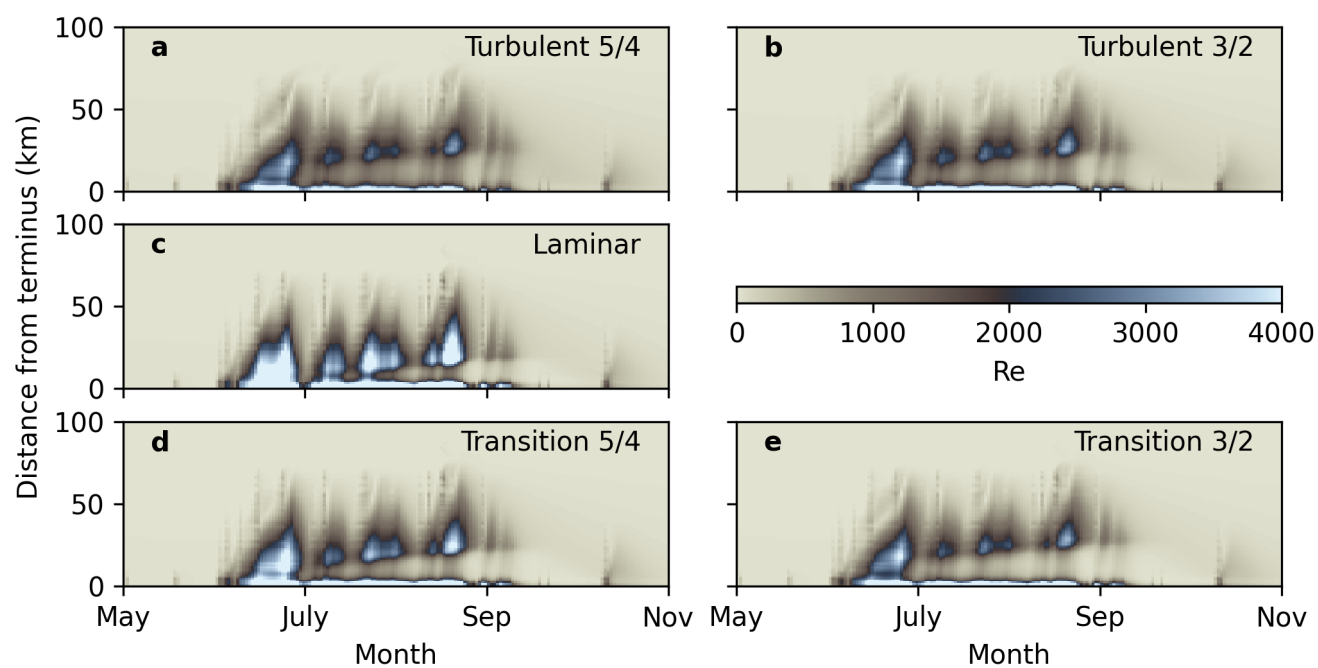


Fig. S15. Seasonal evolution of width-averaged Reynolds number for “low” scenario parameters as in Table S3 for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d), and Transition 3/2 (e) models.

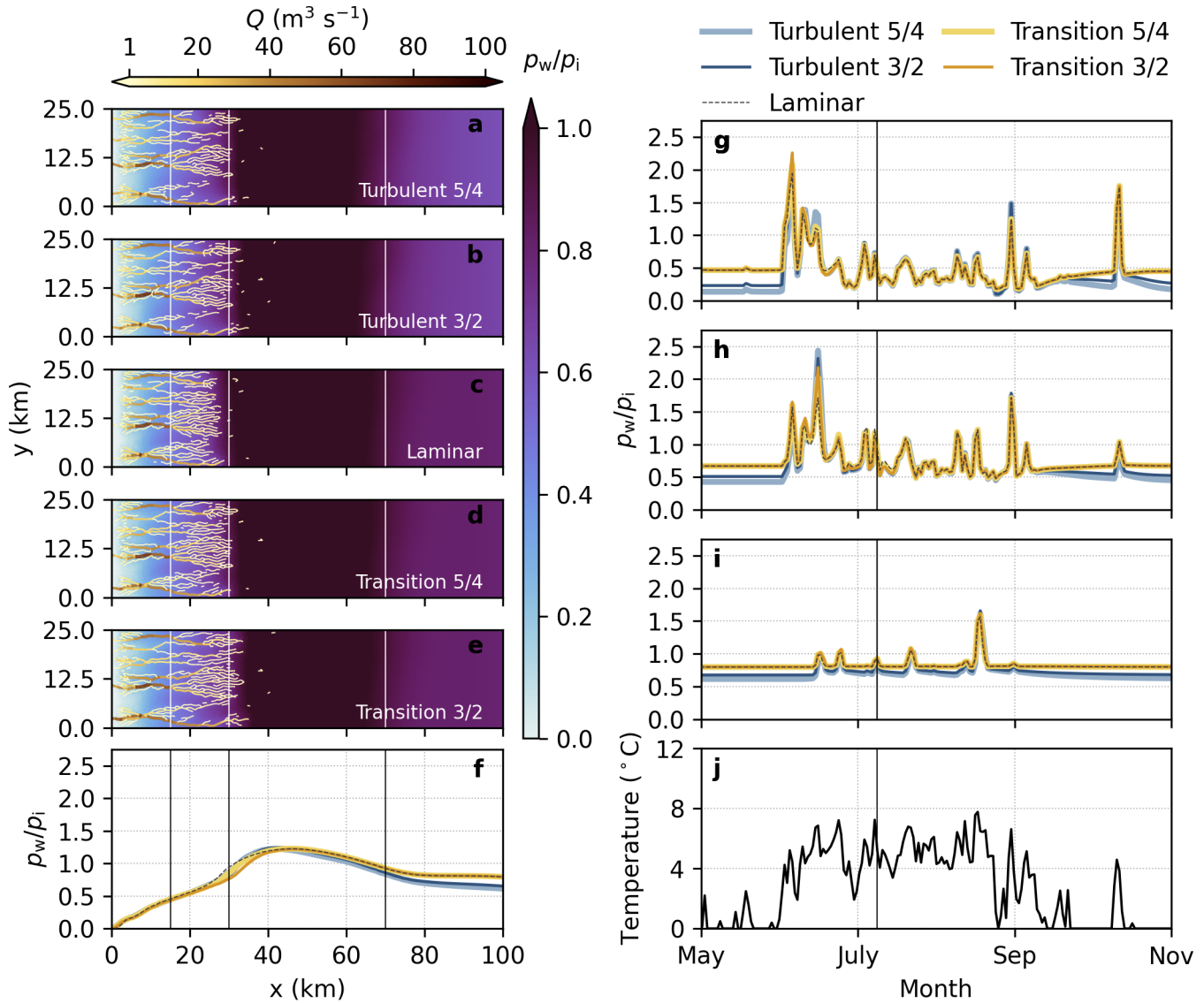


Fig. S16. KAN forcing scenario with reduced englacial storage parameter ($e_v = 2 \times 10^{-5}$ instead of the default $e_v = 1 \times 10^{-4}$). Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

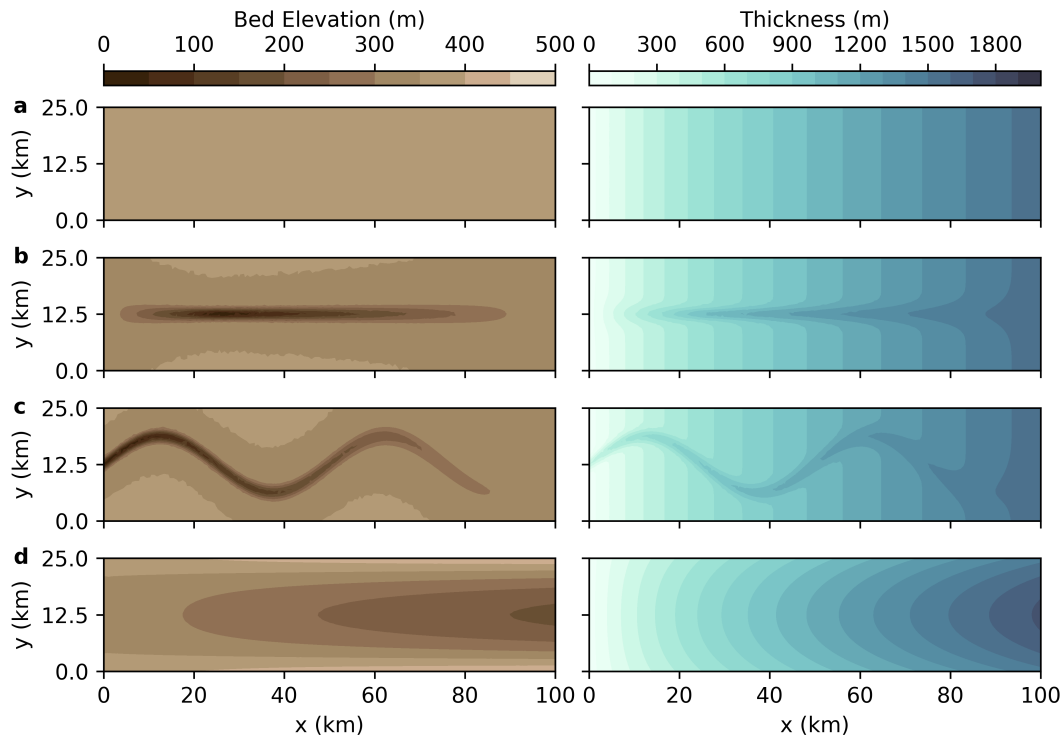


Fig. S17. Bed elevation (left) and ice thickness (right) for flat bed (a), trough (b), sinusoidal (c) and valley (d) scenarios.

228 S3.3 Bed topography

229 The simulations presented so far have used a highly idealized geometry with a flat bed. Since subglacial
 230 topography is expected to be a primary control on the position of subglacial channels in real applications
 231 (e.g., Cook and others, 2020; Dow and others, 2022; Ehrenfeucht and others, 2023), we assess the sensitivity
 232 of our results to three simple, non-flat bed representations (Fig. S17). The “trough” bed has a ~ 6 km-
 233 wide, 350 m-deep trough running down the centre of the domain, intended to represent drainage through
 234 subglacial troughs in west Greenland (Porter and others, 2018). The “sinusoidal” bed extends the bed
 235 trough to reach the terminus and prescribes the trough to follow a sinusoidal path to induce lateral variation.
 236 For this case, the outlet boundary condition is modified to a no-flux condition everywhere except for a
 237 single atmospheric pressure node at the base of the trough. The “valley” bed is U-shaped and with a
 238 retrograde bed sloping downwards from 350 m at the terminus to 200 m at the upper boundary.

239 For the trough (Fig. S18), sinusoidal (Fig. S19) and valley (Fig. S20) beds, the width-averaged floatation
 240 fraction is not significantly affected by the bed topography aside from negative water pressure near the
 241 terminus for the sinusoidal bed as a result of the no-flux boundary condition. The sinusoidal bed produces

242 significantly different spatial patterns of channelization, with channels primarily following the path of the
243 trough.

244 The impact of the valley bed is limited to channels near the domain margin. The marginal channels
245 for the laminar and transition models carry slightly less discharge than in the flat bed case, but this does
246 not noticeably affect width-average floatation fraction.

247 **S3.4 Boundary conditions**

248 To test the sensitivity of our results to the imposed boundary conditions, we modify the terminus boundary
249 condition to a full floatation condition, $p_w = p_i$, and apply synthetic surface melt forcing. Aside from
250 the nearest 5–10 km to the terminus, there is negligible difference in water pressure or channel discharge
251 (Fig. S21). Compared to the atmospheric pressure case, subglacial channels fully terminate slightly further
252 from the margin (Fig. 4).

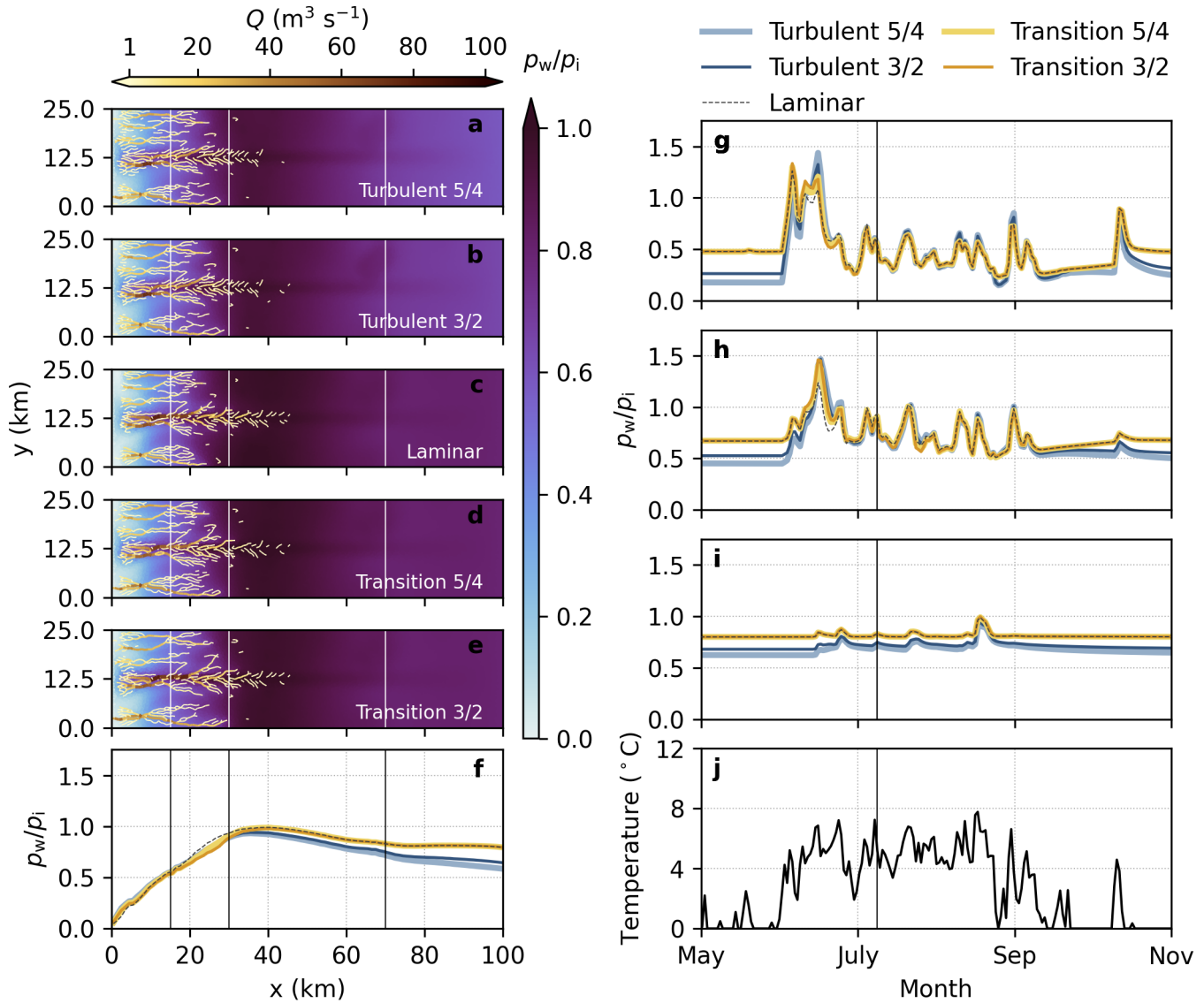


Fig. S18. Trough bed topography with KAN melt forcing. Floataion fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

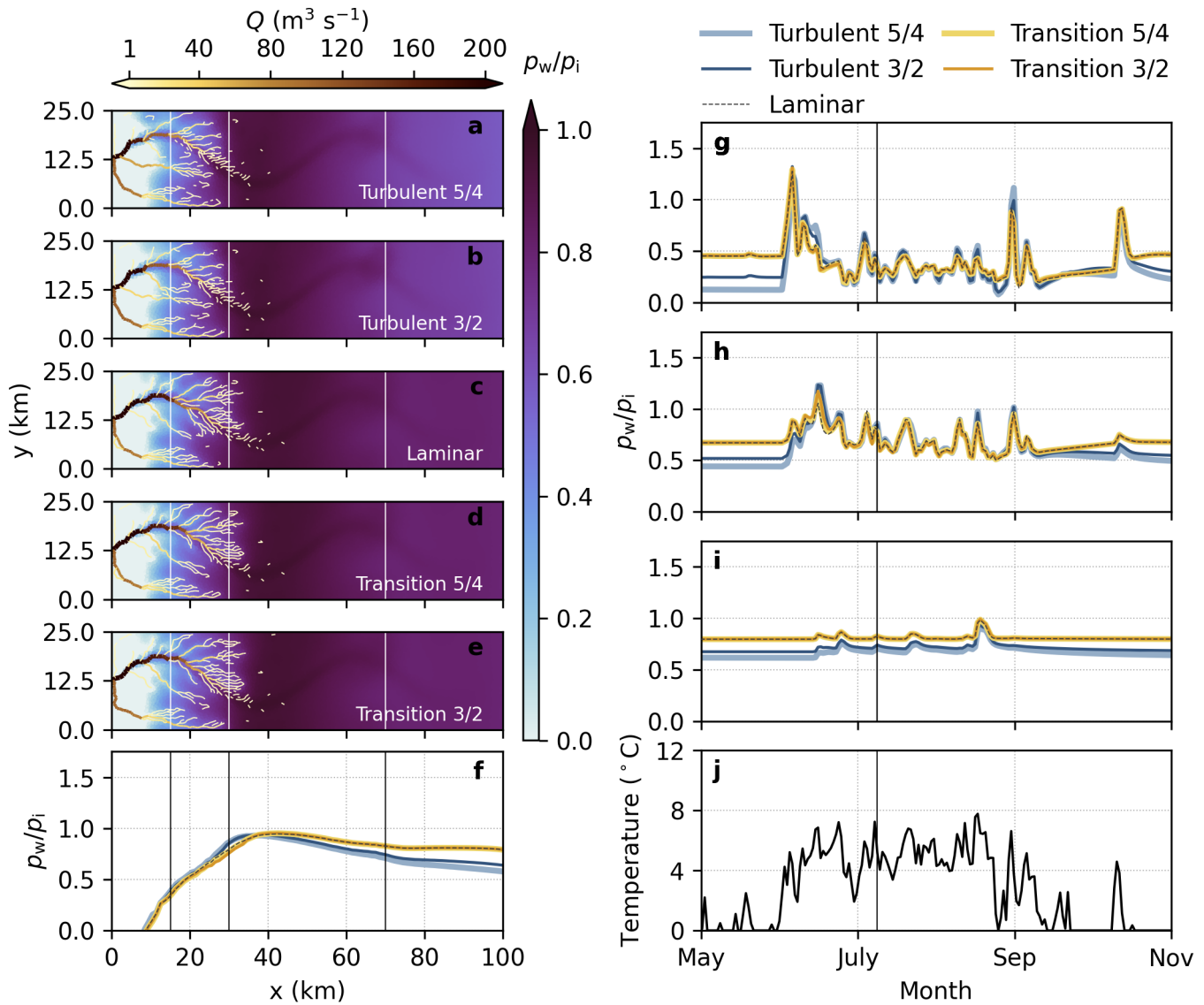


Fig. S19. Sinusoidal bed topography with KAN melt forcing. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

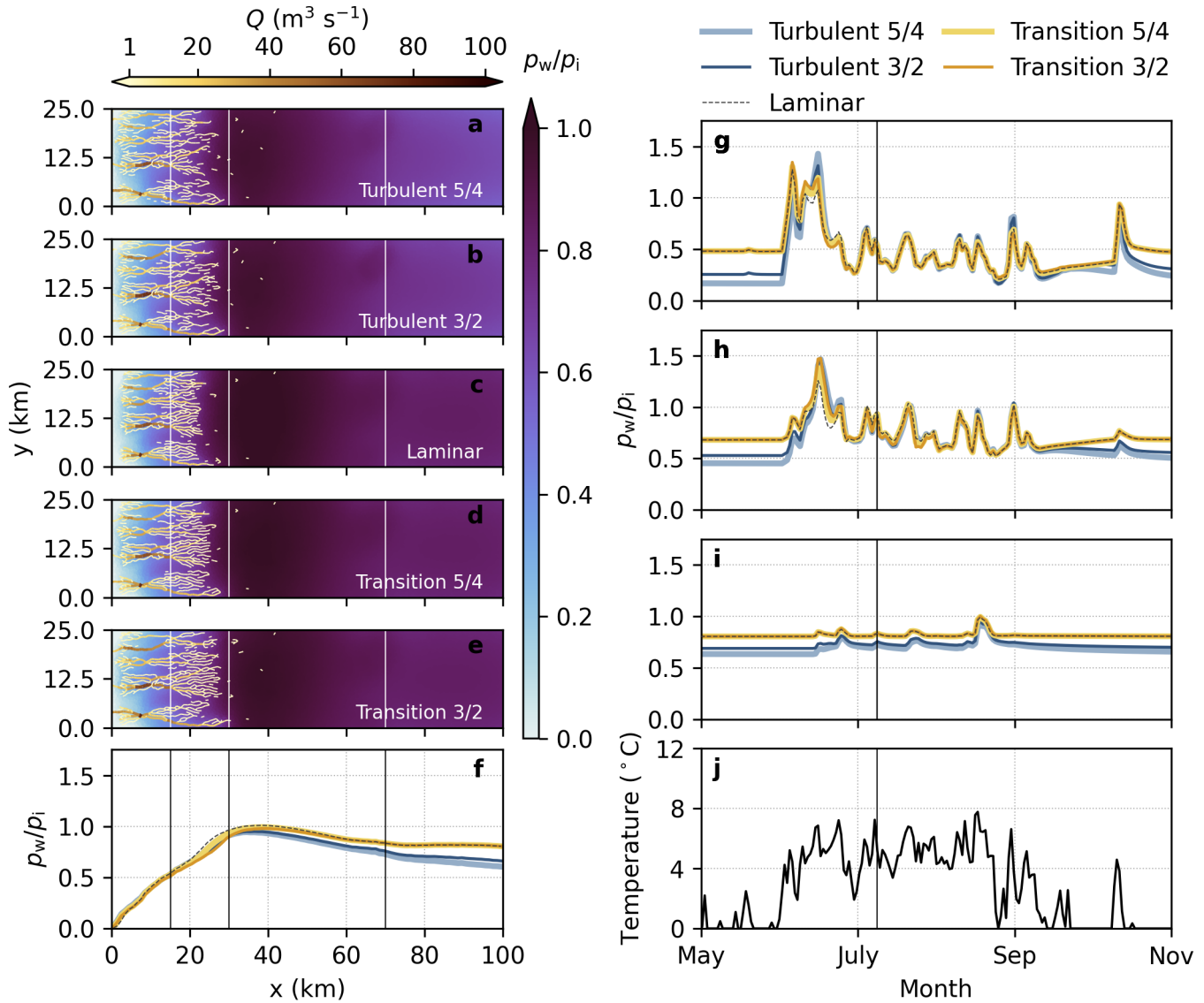


Fig. S20. Valley bed topography with KAN melt forcing. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m a.s.l. used to force the degree-day model (j).

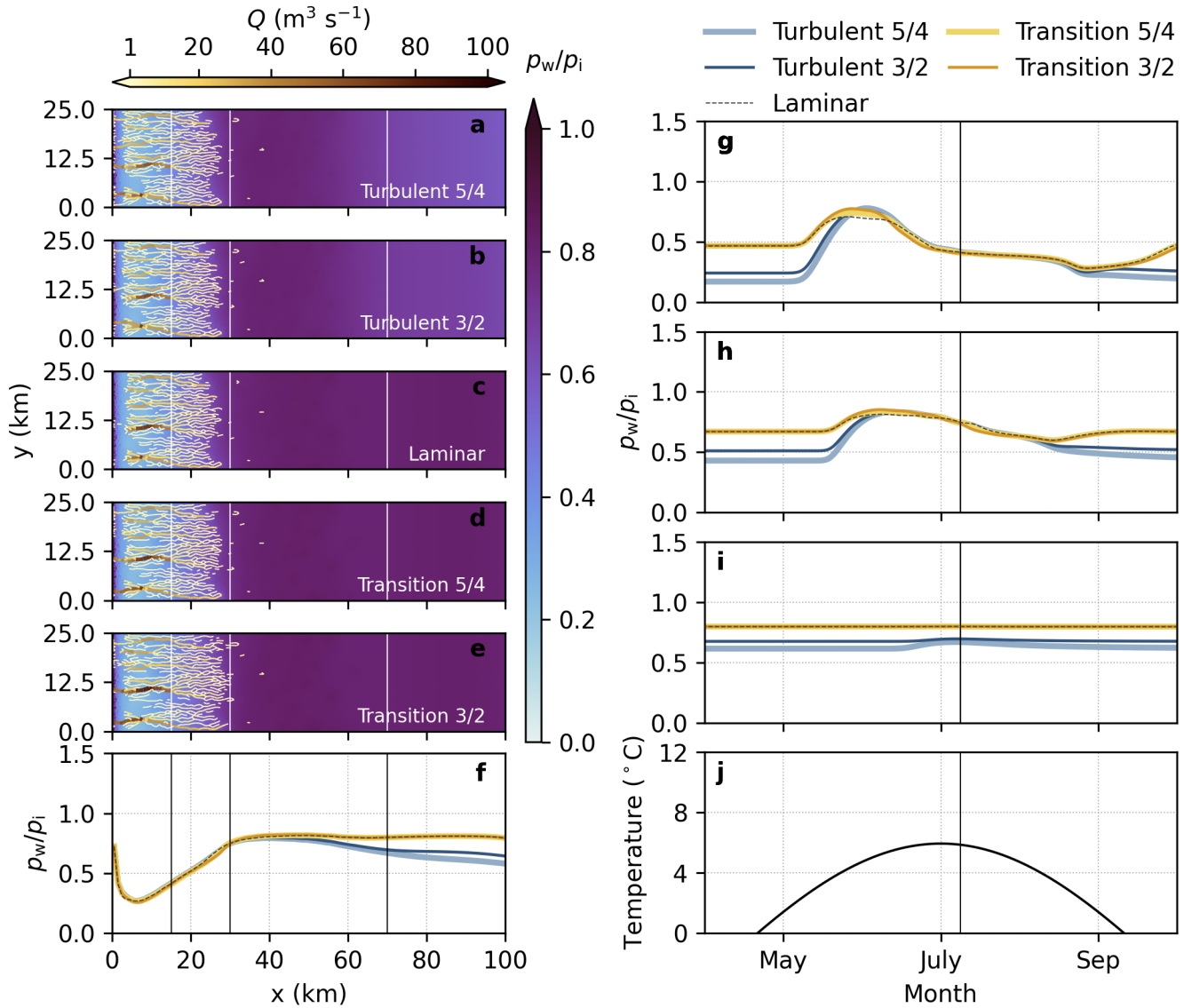


Fig. S21. Floatation boundary condition with synthetic melt forcing. Floatation fraction $\frac{p_w}{p_i}$ and channel discharge on 9 July (a-e) for Turbulent 5/4 (a), Turbulent 3/2 (b), Laminar (c), Transition 5/4 (d) and Transition 3/2 (e) models, and width-averaged floatation fraction on 9 July (f). Width-averaged pressure in bands at $x = 15 \pm 2.5$ km (g), $x = 30 \pm 2.5$ km (h), and $x = 70 \pm 2.5$ km (i) and imposed air temperature at 390 m.a.s.l. used to force the degree-day model (j).

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